

AbsolutAire, Inc.

Engineering Information, Standards & Guidelines

100 cu ft natural gas = 1 therm; 1 therm/hr = 100,000 BTU/hr = 100 MBH

Natural gas cost (Oct 2008 Kalamazoo) = \$1.15/therm

Electricity cost (Oct 2008 Kalamazoo) = \$0.105/KWH (10.5 cents/KWH)

Electric units - Coil sizing requirements: $Kwh = (CFM \times 1.08 \times \Delta T) / 3415$

BTUH = CFM x 1.08 x ΔT (closest to actual) <-> **BTUH** = CFM x 1.14 x ΔT (includes safety)

BTUH = (P x Cp x CFM x 60 x ΔT) / 0.92 (Actual). Where P = Air Density at fan, Cp = 0.241 (constant)

Natural Gas: 27.71" w.c. = 1 PSI = 16 ounces/in² Gas Pressure
 DF burners add humidity at a rate of 11.69grains/lb/100° F temp rise

LP Gas: 93,000 BTU per gallon. LPG cost: approx. \$2.50 per therm

DX Cooling: MBH/12 = Tons of Cooling

Mixed Air: DB mixed air = (DB₁ x %₁) + (DB₂ x %₂) ₁ = Air Stream 1
 Grains/lb (G) mixed air = (G₁ x %₁) + (G₂ x %₂) ₂ = Air Stream 2

Evap Cooling (87% eff): LATdb = EATdb – 0.87(EATdb – EATwb)

ANSI Stds: 100% OA: ANSI Z83.4a, CSA 3.7 (2001) Recirc 80/20 & 85/15: ANSI Z83.18a (2001)
 Spray Bake: ANSI Z83.4 and UL795 Bake Only (Process) & APD90: UL795

Burners:

AA-Series and V-Series Model Units

| | MBH/ft | Natural Gas | | LP Gas | |
|-------------|--------|--------------|------------|--------------|------------|
| | | ΔT°F 100% OA | ΔT°F 80/20 | ΔT°F 100% OA | ΔT°F 80/20 |
| Maxon NP-LE | 750 | 131 | 100 | 95 | 95 |
| Midco HMA2 | 650 | 131 | 100 | 95 | 90 |
| Eclipse | 700 | 143 | 110 | 100 | 95 |

Burners:

R-Series Model Units

| | MBH/ft | Natural Gas | | LP Gas | |
|-------------|--------|--------------|------------|--------------|------------|
| | | ΔT°F 100% OA | ΔT°F 80/20 | ΔT°F 100% OA | ΔT°F 85/15 |
| Maxon NP-LE | 750 | 125 | 100 | 95 | 90 |
| Midco HMA2 | 650 | 125 | 100 | 95 | 90 |
| Eclipse | 700 | 125 | 110 | 95 | 90 |

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|-----------------------------|----------------------|--|
| Motor Drive Losses*: | Up to 1-1/2 HP: 10% | *Fan manufacturers may provide specific values to use in place of these which will supersede these values. |
| | 2 HP to 25 HP: 5% | |
| | 30 HP + higher: 3.5% | |

Power used by electric motors: Watts = Motor Efficiency % x Voltage x Amps

Motor HP = Watts/746 or Kw/0.746 = 2545 BTU

Motor run cost per hour = [(HP x 0.746)/Motor Efficiency] x Electricity Cost per KWH

BHP (approximate) = (Volts x Amps x Power Factor x Efficiency x $\sqrt{3}$)/746

Above based on measured electrical data

Velocity Standards:

| | <u>Max fpm</u> | | <u>Max fpm</u> |
|---------------|----------------|---------------------|-------------------------|
| Inlet Hoods | 600 | Steam/HW Coils | 1000 |
| Evap Media | 500 | HiE Filters | 500 |
| Inlet Plenums | 900 | 30% Pleated Filters | 500 |
| Legs | 600 | DustLok Filters | 450 |
| Cooling Coils | 500 | Alum Filters (2") | 600 (0.10"pd @ 520 fpm) |
| | | Alum Filters (1") | 600 (0.09"pd @ 520 fpm) |

Fan Laws: $CFM_n = CFM_o (RPM_n/RPM_o)$

$SP_n = SP_o (RPM_n/RPM_o)^2$ or $SP_n = SP_o (CFM_n/CFM_o)^2$

$BHP_n = BHP_o (RPM_n/RPM_o)^3$

Friction Losses: Duct: $H_f = 0.0307 \times (V^{0.533}/Q^{0.612}) \times$ Length of Duct in feet

Elbows: Mitered elbow w/o turning vanes = 1.2 V_p

Mitered elbow w/ turning vanes = 0.6 V_p

Velocity: $V(\text{fpm}) = 4005 \times \sqrt{V_p}$ (in. w.c.)

Metric Conversions: $m^3/S = CFM \times 0.000471947$ $mm = \text{inches} \times 25.4$

$W = BTUH \times 0.2931$ $kw = HP/0.7457$

$Pa = \text{in wc} \times 248.84$ $L/s = CFM \times 0.4719$

$KW = (BTUH \times 0.2931)/1000$ $^{\circ}C = (^{\circ}F - 32) \times 5/9$

$kPa = (\text{in wc} \times 248.84)/1000$ $^{\circ}F = (1.8 \times ^{\circ}C) + 32$

$kPa = 6.895 \times PSI$

Also see: www.engineeringtoolbox.com for other formulas and tools